

Power transmission elements

Grooved pulleys for narrow V-belts

Dimensions, materials

DIN**2211**

Part 1

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In keeping with current practice in standards published by the International Organization for Standardization (ISO), a comma has been used throughout as the decimal marker.

See Explanatory notes for connection with International Standard ISO 4183 – 1980 published by the International Organization for Standardization (ISO).

Dimensions in mm

1 Field of application

This standard deals with grooved pulleys for narrow V-belts (hereinafter briefly referred to as pulleys) used in mechanical engineering. The pulleys specified in this standard are used for narrow V-belts conforming to DIN 7753 Part 1. They can also be used for V-belts conforming to DIN 2215 and DIN 2216 (see table 1).

2 Dimensions, designation

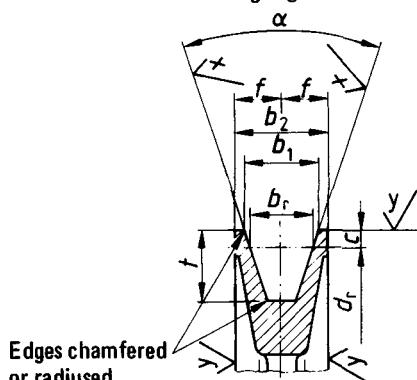
The pulleys need not conform to the illustration; only the dimensions specified shall be adhered to.

General tolerance: DIN 7168 – m

For hub (boss) length l : + IT 14 as defined in DIN 7151, lower deviation 0.

Rim shapes

$$\sqrt{(\sqrt{R_z} 25 \quad \sqrt{R_z} 100)}$$

Single groove

See figure 2 for
run-out tolerances

$$x = \sqrt{R_z 25}$$

$$y = \sqrt{R_z 100}$$

Figure 1.

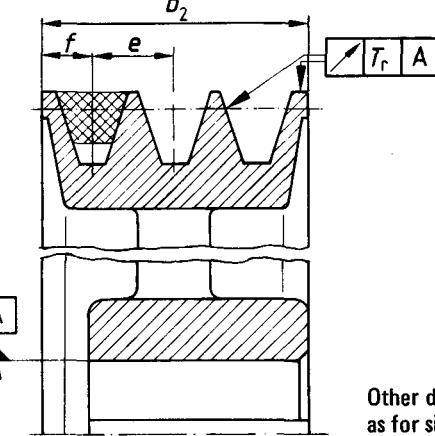
Multi-groove

Figure 2.

Other dimensions and data
as for single groove rim.

Continued on pages 2 to 10

One piece (1T) pulley

Hub always located flush with rim on one side.

Tolerance zones for hub bore:

- H7 for one piece (1T) pulleys;
- U7 for two piece (2T) pulleys.

Other tolerance zones by agreement.

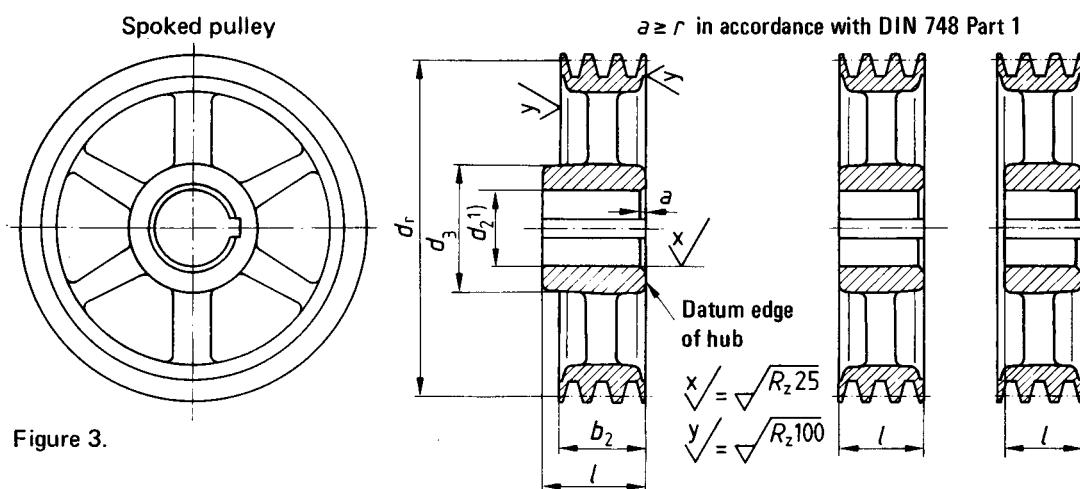


Figure 3.

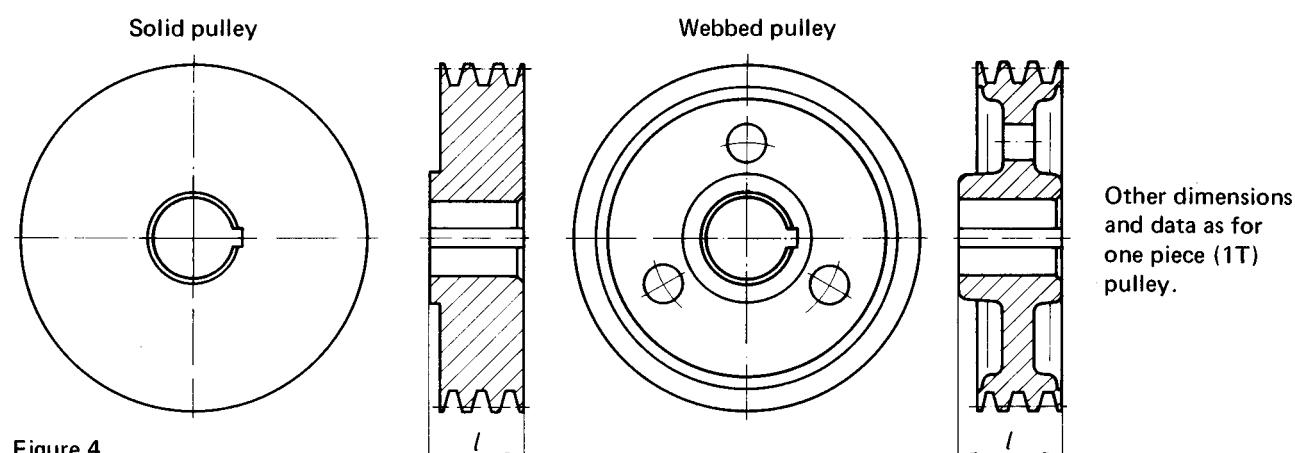
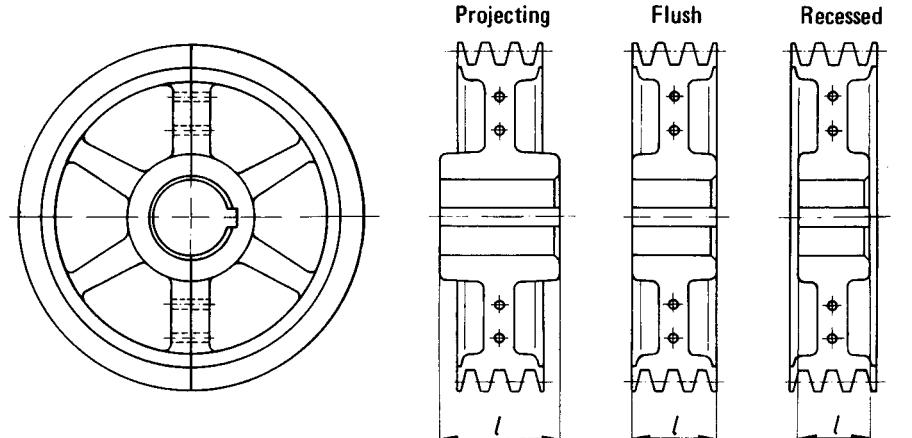


Figure 4.

Two piece (2T) pulley

Hub always located symmetrically to rim.



Other dimensions
and data as for
one piece (1T)
pulley.

Designation of a SPC profile one piece (1T) pulley, with datum diameter $d_r = 500$ mm, number of grooves $z = 8$, hub bore $d_2 = 90$ mm¹⁾, with feather keyway (PN) conforming to DIN 6885 Part 1:

Pulley DIN 2211 – SPC – 1T 500 X 8 X 90 PN

¹⁾ The nominal dimension desired for d_2 shall be stated in the designation.

Table 1.

Narrow V-belt cross sections as in	DIN 7753 Part 1	ISO symbols	SPZ	SPA	SPB	SPC
V-belt cross sections as in	DIN 2215	Symbols	10	13	17	22
	DIN 2216	Symbols	10	13	17	22
Datum width	b_r 2)		8,5	11	14	19
	$b_1 \approx$		9,7	12,7	16,3	22
	c 3)		2	2,8	3,5	4,8
Hub diameter	d_3		$\approx (1,8 \text{ to } 1,6) \cdot d_2$			
Distances between grooves	e 3), 4)		$12 \pm 0,3$	$15 \pm 0,3$	$19 \pm 0,4$	$25,5 \pm 0,5$
	f 3), 7)		$8 \pm 0,6$	$10 \pm 0,6$	$12,5 \pm 0,8$	17 ± 1
Groove depth	t 3)		$11^{+0,6}_0$	$13,8^{+0,6}_0$	$17,5^{+0,6}_0$	$23,8^{+0,6}_0$
α 34° 38°	for datum diameter d_r 5)	≤ 80	≤ 118	≤ 190	≤ 315	
		> 80	> 118	> 190	> 315	
Permissible deviation for $\alpha = 34^\circ$ and 38°		$\pm 1^\circ$	$\pm 1^\circ$	$\pm 1^\circ$	$\pm 30'$	
Rim width b_2 7) $= (z - 1)e + 2f$	for number of grooves z	1	16	20	25	34 6)
		2	28	35	44	59,5 6)
		3	40	50	63	85
		4	52	65	82	110,5
		5	64	80	101	136
		6	76	95	120	161,5
		7	88	110	139	187
		8	100	125	158	212,5
		9	112	140	177	238
		10	124	155	196	263,5
		11	136	170	215	289
		12	148	185	234	314,5

2) The datum width b_r is the reference value for specifying the pulley groove profile. As a rule it is located at the level of the zone of action of the V-belt for which the pulley groove is intended. The datum width has previously been known as the working width (see Explanatory notes).
 3) Calculated on the basis of the decisions made by ISO/TC 41: $c \approx 0,25 b_r$, $e \approx 1,35 b_r$, $f \approx 0,9 b_r$, $t \approx 1,25 b_r$.
 4) The permissible deviation in the distance between grooves for the case of non-neighbouring grooves is double the values stated for e . e can be up to 3 mm greater for sheet metal pulleys and their pairs, and in exceptional cases.
 5) The datum diameter d_r is the diameter associated with the datum width b_r ; it is determinant in the calculation of the transmission ratio. The datum diameter has previously been known as the working diameter (see Explanatory notes).
 6) No hub diameter specified.
 7) Values other than those in table 1 may result for b_2 and f in the case of sheet metal pulleys and their pairs, and in exceptional cases.

