8.3.3 Mechanical Running Test

- **8.3.3.1** The requirements of 8.3.3.1.1 through 8.3.3.1.4 shall be met before the mechanical running test is performed.
- **8.3.3.1.1** All oil pressures, viscosities and temperatures shall be within the range of operating values recommended in the vendor's operating instructions for the specific unit being tested.
- **8.3.3.1.2** Bearings intended to be lubricated by an oil mist systems shall be pre-lubricated.
- **8.3.3.1.3** All joints and connections shall be checked for tightness, and any leaks shall be corrected.
- **8.3.3.1.4** All warning, protective, and control devices used during the test shall be checked and adjusted as required.
- **8.3.3.2** The contract shaft seals and bearings shall be used in the machine for the mechanical running test.
- **8.3.3.3** Shop coupling may be used when testing with shop motors.
- **8.3.3.4** All purchased vibration probes, transducers, and accelerometers shall be in use during the test. If vibration probes are not furnished by the equipment vendor or if the purchased probes are not compatible with shop readout facilities, then shop probes and readouts that meet the accuracy requirements of API Standard 670 shall be used.
- **8.3.3.4.1** Shop test facilities shall include instrumentation with the capability of continuously monitoring and plotting revolutions per minute, and vibration spectra.
- **8.3.3.4.2** The vibration characteristics determined by the use of the instrumentation specified in 8.3.3.4 shall serve as the basis for acceptance or rejection of the machine.
- **8.3.3.4.3** Vibration data shall be recorded in horizontal and vertical directions, at radial planes transverse to each bearing centerline and also in the axial direction as shown in Figure 5.
- 8.3.3.5 All instrumentation used for the tests shall have valid calibration at the time of the test.
- **8.3.3.6** The mechanical running test of the equipment shall be conducted as specified in 8.3.3.6.1 through 8.3.3.6.4.
- **8.3.3.6.1** The mechanical running test shall verify that the operating speed range is free of critical speeds.
- **8.3.3.6.2** A mechanical run test may be performed either before or following the performance test.
- **8.3.3.6.3** The mechanical run test shall be one hour or until bearing temperatures have stabilized; that is, when temperature rise relative to ambient temperature is not more than 2 °F (1 °C) over a 10 minute period. Correct operation of the control system shall be demonstrated, when applicable.
- [•] a) If specified, the LRC/VP shall be mechanically run for four hours. This test shall be performed at the rated flow.
 - b) Mechanical run test shall be conducted with ring liquid present.
 - NOTE The mechanical run test is typically done at either the normal point or rated point for the equipment, as agreed by the vendor and purchaser.
 - **8.3.3.6.4** In the case of adjustable speed units, the test program shall be agreed upon by the purchaser and the vendor, and it shall include a period of operation at maximum continuous speed.

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- **8.3.3.7** The requirements of 8.3.3.7.1 to 8.3.3.7.3 shall be met during the mechanical running test.
- **8.3.3.7.1** The measured unfiltered vibration shall not exceed the limits of 6.8.2.1 and shall be recorded throughout the operating speed range.
- NOTE Typically, the instrumentation is verified for proper operation before and after the mechanical running test.
- **8.3.3.7.2** While the equipment is operating at maximum continuous speed, and at any other speeds or load, or both, that may have been specified in the test agenda, vibration data shall be acquired to determine amplitudes at frequencies other than synchronous.
- **8.3.3.7.2.1** As a minimum, this data shall cover a frequency range from 10 Hz to 1000 Hz. Equipment operating at shaft speeds from 750 rpm down to 300 rpm should be monitored in a frequency range from 5 Hz to 250 Hz.
- **8.3.3.7.2.2** If the amplitude of any discrete, nonsynchronous vibration exceeds 20 percent of the allowable vibration as defined in 6.8.2.1 the purchaser and the vendor shall agree on requirements for further investigation which may include additional testing and on the equipment's acceptability.
- [•] 8.3.3.7.3 If specified, all real-time vibration data as agreed by the purchaser and vendor shall be recorded and a copy provided to the purchaser.
 - **8.3.3.8** If replacement or modification of bearings or seals or dismantling of the case to replace or modify other parts or assembly is required to correct mechanical or performance deficiencies, the initial test will not be acceptable, and the final shop tests shall be run after these deficiencies are corrected.
 - **8.3.3.9** When spare rotors are ordered, each spare rotor shall be manufactured and balanced, in accordance with the requirements of this standard.
- [•] 8.3.3.10 If specified, spare rotors shall be given a mechanical run test or performance test, or both.
 - **8.3.3.11** The purchaser shall advise additional testing requirements for spare parts.

8.3.4 Gas Leak Test

- **8.3.4.1** Each completely assembled LRC/VP casing shall be tested as specified in 8.3.4.2.
- **8.3.4.2** The casing (with the end seals installed) shall be pressurized to the rated discharge pressure, or in the case of vacuum pumps to 1 bar (15 psi), held at this pressure for a minimum of 30 minutes, and subjected to a soap-bubble test or another approved test to check for gas leaks.

8.3.5 Performance Test

- **8.3.5.1** LRC/VP shall be tested in accordance with HEI *Performance Standards for Liquid Ring Vacuum Pumps*. PNEUROP 6612 may be used for a testing standard if agreed.
- [•] 8.3.5.2 The LRC/VP shall be performance tested together with its ring liquid system. The extent of the test, and the applicable test methods, shall be agreed.
 - **8.3.5.3** The machine shall be tested on air and water.
 - **8.3.5.4** For vacuum pumps, test data shall be recorded at one speed at five suction pressures varying from atmospheric pressures to maximum vacuum.
 - **8.3.5.4.1** For compressors, test data shall be recorded at one speed from minimum discharge pressure to maximum discharge pressure.
 - **8.3.5.4.2** The points are subject to negotiation between purchaser and vendor.

8.3.5.5 The dry air and water performance shall be within the tolerances given in Table 11.

Table 11—Performance Tolerances

Variable	Tolerance (%)
Rated inlet volume flow	-0
Rated power	+4
Ring liquid flow rate	±10

- **8.3.5.6** Performance at the certified point shall be calculated from test data in accordance with the vendor's standard procedures, or as otherwise specified. Where the test is to be performed under different conditions or with different fluids from those specified, the method of converting the test results to the specified conditions shall also be agreed upon (see Annex F).
- **8.3.5.7** If it is necessary to dismantle an LRC/VP for a correction, such as improvement of efficiency, the initial test will not be acceptable, and the final hydrostatic test, gas leak test, and performance test shall be repeated after the correction is made.
- **8.3.5.8** The performance test shall be conducted using only one contract rotor.
- **8.3.5.9** The vendor shall maintain a complete, detailed log of all final tests and shall prepare the required number of copies, including test curves and data, certified for correctness. All preliminary tests and mechanical checks shall be completed by the vendor before the purchaser's witnessed performance test.
- **8.3.5.10** The requirements of 8.3.5.10.1 through 8.3.5.10.5 shall be met before the performance test is performed.
- **8.3.5.10.1** The contract shaft seals and bearings shall be used in the machine for the performance test.
- a) The vendor may propose substitute seals for reasons such as incompatibility of the job seals with the test fluid.
- b) Use of substitute seals requires purchaser approval.
- c) The acceptable level of leakage during testing shall be agreed upon by the purchaser and the vendor.
- **8.3.5.10.2** All lubricating-oil and liquid-sealant pressures, viscosities, and temperatures shall be within the range of operating values recommended in the vendor's operating instructions for the specified unit being tested.
- **8.3.5.10.3** Bearings used in oil mist lubrication systems shall be pre-lubricated.
- 8.3.5.10.4 All joints and connections shall be checked for tightness, and any leaks shall be corrected.
- **8.3.5.10.5** All warning, protective, and control devices used during the test shall be checked, and adjustments shall be made as required.

8.3.6 Optional Tests

If specified, the shop tests described in 8.3.6.1 through 8.3.6.7 shall be performed. Test details shall be agreed upon by the purchaser and the vendor.

8.3.6.1 Complete Unit Test

Such components as LRC/VP, couplings, gears, drivers, and auxiliaries that make up a complete unit shall be tested together during the mechanical running test. If agreed by the Purchaser, the complete unit test may be performed in place of separate tests of individual components.

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8.3.6.2 Gear Test

If an external gearbox is provided in the drive train, it shall be tested with the machine unit during the mechanical running test.

8.3.6.3 Helium Test

- **8.3.6.3.1** Pressure containing parts, such as casings, shall be tested for gas leakage with helium at the maximum allowable working pressure.
- **8.3.6.3.2** If the test is conducted with the casing submerged in water, the water shall be at a higher temperature than the nil ductility transition temperature for the material of which the part is made.
- **8.3.6.3.3** The maximum allowable working pressure shall be maintained for a minimum of 30 minutes, with no bubbles permitted.
- **8.3.6.3.4** As an alternative, a non-submerged soap-bubble test or other approved method to check for gas leakage may be performed if approved by the purchaser.
- NOTE A helium test can be appropriate when the molar mass of the gas to be handled is less than 12 or if the gas contains more than 0.1 mole percent hydrogen sulfide.

8.3.6.4 Sound-level Test

The sound-level test shall be performed in accordance with ISO 3744 or another agreed standard.

NOTE This test usually does not reflect field sound levels due to shop test environment.

8.3.6.5 Ring Stability Test

For adjustable speed machines a test agreed to between the purchaser and the vendor shall be conducted to establish the operating limits of liquid ring stability.

8.3.6.6 Auxiliary Equipment Test

Auxiliary equipment such as oil systems, gears and control systems shall be tested in the vendor's shop. Details of the auxiliary equipment tests shall be developed jointly by the purchaser and the vendor.

8.3.6.7 Spare Parts Test

Spare parts such as couplings, gears, diaphragms, bearings, and seals shall be tested.

8.4 Preparation for Shipment

- [•] 8.4.1 Equipment shall be suitably prepared for the type of shipment specified.
 - **8.4.1.1** Blocked rotors shall be identified by means of corrosion-resistant tags attached with stainless steel wire.
 - **8.4.1.2** The preparation shall make the equipment capable of withstanding six months of outdoor storage from the time of shipment, with no disassembly required before operation, except for inspection of bearings and seals.
 - **8.4.1.3** If storage for a longer period is contemplated, the purchaser will consult with the vendor regarding the recommended procedures to be followed.
 - **8.4.2** The equipment shall be prepared for shipment after all testing and inspection have been completed and the equipment has been released by the purchaser.

- **8.4.3** Except for machined surfaces, all exterior surfaces that may corrode during shipment, storage, or in service, shall be given at least one coat of the manufacturer's standard paint. The paint shall not contain lead or chromates.
- NOTE Austenitic stainless steels are typically not painted.
- **8.4.4** Exposed shafts and shaft couplings shall be wrapped with waterproof, moldable waxed cloth, or vapor-phase-inhibitor paper. The seams shall be sealed with oil-proof adhesive tape.
- **8.4.5** Bearing assemblies shall be fully protected from the entry of moisture and dirt.
- **8.4.5.1** If vapor-phase-inhibitor crystals in bags are installed in large cavities to absorb moisture, the bags shall be attached in an accessible area for ease of removal.
- **8.4.5.2** Where applicable, bags shall be installed in wire cages attached to flanged covers, and bag locations shall be indicated by corrosion-resistant tags attached with stainless steel wire.
- **8.4.6** The interior of the equipment shall be clean; free from scale, welding spatter, and foreign objects; and sprayed or flushed with a suitable rust preventative that is water soluble or can be removed with solvent. The rust preventative shall be applied through all openings while the machine is slow-rolled.
- **8.4.7** Flanged openings shall be provided with metal closures at least 5 mm ($^{3}/_{16}$ in.) thick, with elastomer gaskets, and at least four full-diameter bolts. For studded openings, all nuts needed for the intended service shall be used to secure closures.
- **8.4.8** Threaded openings shall be provided with steel caps or round-head steel plugs. In no case shall nonmetallic (such as plastic) caps or plugs be used.
- NOTE These are shipping plugs; permanent plugs are covered in 6.3.14.
- **8.4.9** Openings that have been beveled for welding shall be provided with closures designed to prevent entrance of foreign materials and damage to the bevel.
- **8.4.10** If a spare rotor is purchased, it shall be prepared for unheated indoor storage for a period of at least three years.
- **8.4.10.1** The rotor shall be treated with a rust preventative and shall be housed in a vapor-barrier envelope with a slow-release volatile-corrosion inhibitor.
- [•] 8.4.10.2 The rotor shall be crated for domestic or export shipment, as specified.
 - **8.4.10.3** A purchaser approved resilient material 3.0 mm ($\frac{1}{8}$ in.) thick [not tetrafluoroethylene (TFE) or polytetrafluoroethylene (PTFE)] shall be used between the rotor and the cradle at the support areas.
 - **8.4.10.4** The rotor shall not be supported at journals.
 - **8.4.11** Internal steel areas of bearing housings and carbon steel oil systems' auxiliary equipment such as reservoirs, vessels, and piping shall be coated with suitable oil-soluble rust preventive.
 - 8.4.12 Lifting points and lifting lugs shall be clearly identified on the equipment or equipment package.
 - **8.4.12.1** The recommended lifting arrangement shall be as described in the installation manual.
 - 8.4.12.2 The recommended lifting arrangement shall be identified on boxed equipment.

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- **8.4.13** The equipment shall be identified with item and serial numbers. Material shipped separately shall be identified with securely affixed, corrosion-resistant metal tags indicating the item and serial number of the equipment for which it is intended.
- **8.4.14** The vendor shall provide the purchaser with the instructions necessary to preserve the integrity of the storage preparation after the equipment arrives at the job site and before startup, as described API RP 686, Chapter 3, "Recommended Practices for Machinery Installation and Installation Design".
- **8.4.15** Auxiliary piping connections furnished on the purchased equipment shall be impression stamped or permanently tagged to agree with the vendor's connection table or general arrangement drawing. Service and connection designations shall be indicated.
- **8.4.16** The fit-up and assembly of machine-mounted piping, intercoolers etc. shall be completed in the vendor's shop prior to shipment.
- 8.4.17 Wood used in export shipping shall comply with the requirements of ISPM Publication No.15 [2].
- **8.4.18** Composition wood product such as Particleboard, Medium Density Fiberboard (MDF), and Oriented Strand Board (OSB) shall not be used.

8.4.19 Package Markings and Shipping Documentation

- [•] 8.4.19.1 All markings shall be in English and other specified language.
 - **8.4.19.2** Package markings shall be stenciled on two opposite sides of the shipping unit. A shipping unit may be a box, carton, bundle, crate, drum, loose self-supported piece of equipment etc.
 - **8.4.19.3** Lettering shall be between 76 mm to 125 mm (3 in. to 5 in.) high in weatherproof black ink to ensure visibility.
 - **8.4.19.4** Shipping packages that cannot be stenciled directly shall have attached corrosion resistant metal tags with raised markings.
 - **8.4.19.5** Shipping packages shall be marked with industry standard cautionary symbols indicating center of gravity, sling or lifting points, top heavy packages, fragile and liquid contents, moisture sensitive contents etc. per ASTM D5445-05, *Standard Practice for Pictorial Markings for Handling of Goods*.
 - **8.4.19.6** Package markings shall include:
 - a) purchaser's purchase order number and tag number
 - b) shipping unit piece number
 - c) gross weight
 - d) dimensions
 - e) purchaser's project name
 - **8.4.19.7** Packaged equipment shall be shipped with duplicate packing lists—one inside and the other on the outside of the shipping container. Also, a paper copy of package markings shall be inside each container.
 - **8.4.19.8** One copy of the manufacturer's installation instructions shall be packed and shipped with the equipment.
 - **8.4.19.9** Equipment or materials that contain or are coated with chemical substances shall be prominently tagged at openings to indicate the nature of contents and precautions for shipping, storage, and handling.

- NOTE Some examples include oils, corrosion inhibitors, antifreeze solutions, desiccants, hydrocarbon substances, and unused paint.
- 8.4.19.9.1 Substances that are supplied with the shipment shall have a Safety Data Sheet (SDS).
- **8.4.19.9.2** If a substance is exempt from regulation, a statement to that effect shall be included.
- **8.4.19.9.3** At least two weeks before shipment, SDSs shall be forwarded to the receiving facility, to allow planning for handling of any regulated substances.
- 8.4.19.9.4 SDSs in protective envelopes shall be affixed to the outside of the shipping package.

9 Vendor's Data

- **9.1** The purchaser may specify the content of proposals, meeting frequency and vendor data content/format identified in Annex B. Annex B provides a general outline of information that potentially may be requested by the purchaser.
- [•] 9.2 If specified, the information specified in Annex B shall be provided.

Annex A

(informative)

Data Sheets

		LIQUID RING COMPRESSOR AND		JOB NOITEM NOPURCHASE ORDER NO.											
		VACUUM PUMP SYSTEMS (API 681-2nd	4)	SPECIFICATION NO. REVISION NO. DATE											
d		DATA SHEETS	,												
REV		SI UNITS (kPa)		PAGE	1 OF	13 BY									
H	4 [APPLICABLE TO: O PROPOSAL O PURCHASE	O AS BUIL												
Н	-	FOR		UNIT											
П	_	SITE		DRIVEN EQU	IP. NO. REQ	UIRED									
П	4	SERVICE		SIZE AND TY	PE										
	5	MANUFACTURER		SERIAL NO.											
	6														
Ш	_	NOTE: O INDICATES INFORMATION TO BE COMPLETED BY F		☐ BY I	MANUFACTU	RER 🖸	BY MANUFAC	CTURER OR F	PURCHASER						
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П	15														
	16		SYSTEM	DESCRIPTIO	N										
	17	DRIVEN EQUIPMENT TYPE		SYSTEM	TYPE (ANNE	X E)									
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Н	_	SERVICE (6.1.1)		PARTIAL RECIRCULATION (FIGURE E.2) TOTAL RECIRCULATION - VERTICAL SEPARATOR (FIGURE E.3)											
Н	_	O CONTINUOUS O INTERMITTENT (STARTS/DAY): O TRL LEVEL (6.1.1.1): O OTHER					RTICAL SEPAR RIZONTAL SEF								
Н		O VENDOR TO PROVIDE FIELD PROVEN DOCUMENTATION (6.	1 1 2)		AL RECIRCO	LATION - HO	RIZUNTAL SEF	ARATOR (FIG	OURE E.4)						
	23		OCESS OPERA	TING CONDIT	IONS (6.1.2)										
	24														
	25		START-UP	NORMAL	RATED		OTHER	CONDITIONS							
Ш	26	ALL DATA ON PER UNIT BASIS				Α	В	С	D						
Ш	_	O PROCESS CONDITION													
Н	_	O GAS HANDLED (ALSO SEE PAGE 2)					-								
Н	_	O SM³/H (101.325 kPaA & 15°C DRY) O MASS FLOW (kg/h) O WET O DRY													
Н	31	SUCTION CONDITIONS:					<u> </u>								
П	-	O PRESSURE (kPaA)													
	33	O TEMPERATURE (°C)													
	34	O RELATIVE HUMIDITY (%)													
Ш	_	O MOLECULAR WEIGHT													
Ш	_	☐ Cp/Cv ☐ K₁ ☐ K _{AVG}													
Н	_	COMPRESSIBILITY													
Н	_	☐ INLET VOLUME FLOW (m³/h) ☐ WET ☐ DRY DISCHARGE CONDITIONS:					l								
Н	_	PRESSURE (kPaA)													
	_	☐ TEMPERATURE (°C)													
П	_	☐ Cp/Cv ☐ K ₂ ☐ K _{AVG}													
	43	COMPRESSIBILITY Z ₂ Z _{AVG}													
Ш	44	PACKAGE DISCHARGE CONDITIONS:													
Ш	45	O PRESSURE (KPaA)					ļ								
Н	-	TEMPERATURE (°C)					<u> </u>								
Н	\neg	MASS FLOW OF NON-CONDENSIBLES (kg/h)					1		 						
Н	48 49	MASS FLOW OF SAT LIQUID (AFTER SEPARATOR) (kg/h) O SYSTEM SETTLE OUT PRESSURE (kPaA)													
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П	_	POWER REQUIRED (ALL LOSSES INCL) (kW)													
		SPEED (RPM)													
	53	VOLUMETRIC EFFICIENCY (%)													
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l l	55														
П	-	—													
	-	PERFORMANCE CURVE NO.													

	LIQUID DING OCCUPATION AND									JOB NOITEM NO										
	LIQUID RING COMPRESSOR AND								PURCHASE ORDER NO.											
	VACUUM PUMP SYSTEMS (API 681-2nd)									SPECIFICATION NO.										
>			DATA:	SHEET	S				REVISION NO. DATE											
REV			SI UNI	TS (kP	a)				PAGE	2	OF	13 E	3Y							
Н	1					PRO	CESS OF	PERATING	PAGE _ 2											
H	2	GAS ANALYSIS (6.	1 3)	START	NOR-	RATED			ONDITIONS											
H		O MOL % O		UP	MAL	IVAILD	A	В	С	D	1									
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H		NITROGEN	28.016		-		-			+										
Н	_	WATER VAPOR	18.016							+										
Н		CARBON MONOXIDE	28.010							+	-									
Н	_	CARBON DIOXIDE	44.010		-					+	<u> </u>									
\vdash	_	HYDROGEN SULFIDE	34.076		_					+	(6.10.1	1 11)								
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\vdash		METHANE	16.042							+	(0.10.	1.0)								
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Н		ETHANE	30.068		-					+										
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Н		I-PENTANE	72.146							+										
H	_	n-PENTANE	72.146		-	1				+										
Н		HEXANE PLUS	72.140							+										
H	24	HEXAINE PLUS	1		-	1	-			+										
H	25		 		-	1	-			+										
H	26		1		-	1	-			+										
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	33						LRC/VP	OUTLET	MATER	RIALS BAL	ANCE									
Н	34																			
П	35						N	IASS FLO	OW MW SG TEMP							\neg				
	36	AT CERTIFIED POINT:						(kg/h)						(°F)	PRESSURE (psia)					
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П	38	CONDENSED LIQUID	c c										\top			┪				
	39			BONS												7				
	40	_														7				
	41	_											_							
	42					R	ING LIQU	ID OPER	ATING C	CONDITION	NS (6.1.	.4)								
П	43	LIQUID TYPE								CORRO	SIVE/EI	ROSIVE AGE	NT							
	44 MIN NORMAL MAX RATED											NCENTRATIC			(ppm)					
	45	SUPPLY PRESSURE	(kPa									RATION		(ppm)						
		SUPPLY TEMPERATI										FLAMMAE				(6.1.6)				
		VAPOR PRESSURE	(kPa									— URGE SYSTE								
П		VISCOSITY						\neg		_					(m³/h)				
П		SPECIFIC HEAT								CIRCUL	ATION	RATE				m³/h)				
П	_	SPECIFIC GRAVITY										GN PRESSUR				kPaG)				
П		MINIMUM CONTINUC	US RING	LIQUID F	LOW (3.3	4)	•	(m³/				TY IN RING			,	•				
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		JOB NO. ITEM NO.										
	LIQUID RING COMPRESSOR AND	PURCHASE ORDER NO.										
	VACUUM PUMP SYSTEMS (API 681-2nd)	SPECIFICATION NO.										
	DATA SHEETS	REVISION NO. DATE										
REV	SI UNITS (kPa)	PAGE 3 OF 13 BY										
屵	` '											
Ш		INE CONSTRUCTION FEATURES										
Ш	2 SPEEDS	ROTORS										
Ш	3 ROTATION, VIEWED FROM COMP. DRIVEN END: CW CCW	NO. SOLID HOLLOW										
11	4 MAXIMUM CONTINUOUS SPEED (3.28) RPM	DIAMETERS										
П	5 TRIP (3.64) RPM	■ NO. VANES PER ROTOR										
П	6 MAX. TIP SPEEDS: (m/s) @ MAX. CONT. SPEED	TYPE OPEN CLOSED OTHER										
П	7 MINIMUM CONTINUOUS SPEED (3.32) RPM	FABRICATION METHOD										
Н	8 LATERAL CRITICAL SPEEDS: (6.8.1.1)	MATERIAL										
Н	9 FIRST CRITICAL (DRY) RPM	MAX YIELD STRENGTH										
Н	10 SECOND CRITICAL (DRY)	BRINNEL HARDNESS MAX MIN										
Н	11 LATERAL CRITICAL SPEED - BASIS:	O DYNAMIC ROTOR BALANCING (6.6.5)										
Н		1 =										
Н	12 O TYPICAL LATERAL ANALYSIS REQUIRED (6.8.1.5)	ISO 21940-11 G2.5 (6.6.5) ISO 21940-11 G1 (6W/n) (6.6.6)										
Н	13 O UNDAMPED STIFFNESS MAP REQUIRED	2017										
Н	14 O OTHER TYPE ANALYSIS:	SHAFT										
Ш	15 TORSIONAL CRITICAL SPEEDS: (6.8.1.7)	MATERIAL										
Ш	16 FIRST CRITICAL RPM	DIA @ ROTOR (mm) DIA @ COUPLING (mm)										
Ш	17 SECOND CRITICAL RPM	SHAFT END: TAPERED (7.2.8) CYLINDRICAL (7.2.7)										
Ш	18 O TRAIN TORSIONAL ANALYSIS REQUIRED (6.8.1.9)	SHAFT SLEEVES (6.6.8):										
11	19 O OTHER TYPE ANALYSIS:	AT SHAFT SEALS MATL.										
П	20											
П	21 CASING	BEARINGS AND LUBRICATION										
П	22 MODEL	BEARING HOUSING (6.9.2):										
П	23 CASING SPLIT	MATERIAL:										
Н	24 MATERIAL (6.10.1.2) CLADDING	CAST IRON DUCTILE IRON STEEL										
Н	25 THICKNESS (mm) CORR. ALLOW (mm)	SEAL TYPE: LABYRINTH (6.9.2.1)										
Н	26 MANIFOLD MATERIAL	OTHER (6.9.2.1.1)										
H	27 PORT PLATE/CONE MATERIAL	BEARING TYPE:										
	28 MAX. ALLOWABLE WORKING PRESS. (3.27) (kPaG)	TYPE NO. CLEARANCE										
Н	29 MAX. DESIGN PRESS. (3.9) (kPaG)	RADIAL ROS SEEARANGE										
Н	29 NAX. DESIGN PRESS. (S.9) (KPaG)											
Н	30 TEST PRESSURE: HELIUM (8.3.6.3) (KPaG) 31 HYDRO (8.3.2) (KPaG)	THRUST										
Н	 ' '											
Н	32 MAX. ALLOWABLE TEMPERATURE (3.26)	LUBRICATION (6.9.3)										
	33 MINIMUM OPERATING TEMPERATURE (°C)	GREASE (6.9.3.1) FLOOD RING OIL FLINGER										
Н	34 O RELIEF VALVE SETTING (6.2.3)(kPaG)	PRESSURE PURGE OIL MIST (6.9.2.5.2) PURE OIL MIST (6.9.2.5.1)										
Н	35 O MARGIN FOR ACCUMULATION(kPaG)	CONSTANT LEVEL OILER (6.9.2.4.2)										
Н	36 HEXAGONAL CAP SCREWS REQ'D FOR ASSEMBLY (6.2.7)	VENTED TO BEARING HOUSING (6.9.2.3.7)										
Ш	37 O CASING VERTICAL JACKSCREWS AND DOWEL PILOT HOLES	OIL SUMP COLLECTION CONTAINER (6.9.2.3.8)										
	38 RADIOGRAPH QUALITY	O BEARING HOUSING FOR FLOODED LUBE OIL REQUIRED (6.9.2.4)										
Ш	39 THREADED OPENINGS FOR TAPERED PIPE PER ISO 7-1 (6.3.10)	O OIL VISCOSITY ISO GRADE										
Ш	40	O OIL HEATER O ELECTRIC O STEAM										
Ш	41	O OIL PRESSURE TO BE GREATER THAN COOLANT PRESSURE										
	42											
	43	VIBRATION DETECTORS (7.6.4.6)										
П	44	O TYPE										
П	45	MFR										
П	46	O NO AT EACH SHAFT BEARING:										
П	47	MONITOR SUPPLIED BY (7.6.4.6.2): VENDOR CUSTOMER										
П	48	LOCATION ENCLOSURE										
П	49	MFR MODEL										
Н	50	SCALE RANGE ALARM - SET @ (mm/s)										
H	51	SHUTDOWN - SET @ (mm/s) O TIME DELAY: (sec)										
H	52	(300)										
H	53 REMARKS:											
Н	54											
\vdash	55											
Н	56											
\vdash												

	LIQUID RING COMPRESSOR AND								JOB NOITEM NO PURCHASE ORDER NO											
			VACUUM PI			-	l 681-2	2nd)		SPECIFICATION NO.										
REV					SHEET					REVISI	ON NO.			DATE						
쮼	SI UNITS (kPa)					PAGE	4	OF	13	BY										
Г	1								C	CONNECTIONS										
	2													FLANG	ED STUD	DED OR	MATING	RING &	GASKET	
	3		CONNECTION	N	Ο.	SIZE		FACING		RATING		POSIT	ION		READED (Y VEND		
	4		SYSTEM INLET																	
	5		SYSTEM DISCHARGE																	
	6	5	SYSTEM FILL																	
		TEM	SYSTEM MAKE-UP																	
	8	SYS	SYSTEM DRAIN																	
	9		CW INLET																	
L	10		CW OUTLET																	
H	11		VENTS						_											
H		=	MACHINE SUCTION	_																
\vdash	13	ų,	MACHINE DISCHARGE	E																
	14 15	_	RING LIQUID INLET																	
\vdash	16	_	DRAINS			ALLOWE	D FORCE	ES AND I	MOMENT	ON SYST	EM AND	L IOUID E	RING MAC	HINE (6	5)					
	17		POSITION	evete	M INLET		A DISCH.		EM FILL	_	AKE-UP	_	M DRAIN	· •	INLET	CWC	UTLET	\/E	NTS	
	18	5	LOAD		MOMENT	FORCE			MOMEN.		MOMEN.		MOMENT		MOMENT		_	FORCE		
	19	STEM	AXIAL																	
	20	<u>~</u>	VERTICAL																	
	21		HORZONTAL 90°																	
	22	Ē	POSITION	CASING	SUCT. 1	CASING	SUCT. 2	DISCH	IARGE 1	DISCH	DISCHARGE 2 RING LIQ. INLET				T DRAINS			UNITS		
	23	튄	AXIAL														FORCE (kN)			
	24	MAC	VERTICAL													1	MOMEN.	T (N-m)		
	25		HORZONTAL 90°																	
	26									(7.2) AND	GUARD	S (7.3)								
			COUPLING- TYPE:	KE)		O TAP		O STE	RAIGHT	BAL	ANCE T	o: O	AGMA 9	000 CL. 9	7.2.2)	O ISO	21940-11	G6.3 (7	.2.2 f)	
	-		DIRECT COUPLED (SI	EALLESS	CONST	RUCTION	1)					_	AGMA 9							
			MFR/MODEL						_	1		TACHED			EET					
L			FURNISHED BY:			~ · ·			_			ATING RE								
			SPACER LENGTH LIMITED END FLOAT I	DECLUDE	- D	RAT	ING:			- _	_	N-LUBE	_	EASE	OTH	HER				
	-		_	LY ENCL			SEMI-OF	DEN				RIVE (7.4) YPE (7.4.1								
	34	GU			.USED ING (7.3.7	_	SEIVII-OF	ZEIN		- 1-		IANUFAC								
		0	FURNISHED BY:	V-SFAICK	1140 (7.5.1	')				1 -		ERVICE F		7 4 1 4)						
\vdash			VENDOR MOUNT HAL	F COUP	ING (7.2.	12)									BALANCE	(7.4.5.7)				
	37	Ť							ВА	SEPLATE						, ,				
	38		BASEPLATE FURNISH	HED BY (7.5.1):							COLUM	и моинт	ING (7.5.	.20)					
	39	МО	UNTED EQUIPMENT (7	7.5.16):	O MAG	CHINE	O DRI	VER O	COMPL	ETE SYST	ем С	NON-SK	ID DECK	NG (7.5.2	26)					
	40	0	EPOXY GROUT TYPE	FOR FIE	LD INSTA	LL. (7.5.7	'):					STAINLE	ESS STEE	EL SHIM	THICKNES	SS:		QTY		
			LEVELING PADS (7.5.	18)			(7.5.10)	D	SPACE	R PLATE I	NSTALL	ED UNDE	R EQUIPM	MENT FE	ET (7.5.10).6)				
	42				WEIGHT	S (kg)								SPACE R	EQUIREN	IENTS				
			LIQUID RING MACHIN	ΙE																
		_	DRIVER											l	IGTH .	l	DTH	HEIG		
	_	_	GEARBOX											(r	nm)	(n	nm)	(ft-ir	1)	
L		_	SEPARATOR									RING MA	CHINE						_	
		=	COMPLETE UNIT								DRIVER									
H		18 CRANE FOR MAINTENANCE									SEPAR	ATOR ETE UNIT		<u> </u>					_	
\vdash		49 ADDITIONAL AUX EQUIP. 50 TOTAL SHIPPING									RESTRIC									
\vdash	5U 51									٦	SPACE	KESI KIC	HONS					<u> </u>		
H		B E '	MARKS:																	
	53	_[MANNO.																	
	54																			
	55																			
	56																			
H	57	-																		